## 淡江大學 103 學年度碩士班招生考試試題

系別: 化學工程與材料工程學系 科目: 輸送現象與單元操作

考試日期:3月2日(星期日) 第2節

本試題共4大題,2頁

- 1. A constant density, Newtonian fluid flows steadily between two infinite, parallel plates as shown in Fig. 1. The bottom plate is stationary and the top plate (at z=h) is moving steadily in the x direction with velocity 'A'. The gravity is not important.
  - (1) Please solve the pressure profile with all the <u>assumptions</u> and <u>boundary conditions</u> clearly stated. (15%)
  - (2) Please solve the velocity profile with all the <u>assumptions</u> and <u>boundary conditions</u> clearly stated. (15%)

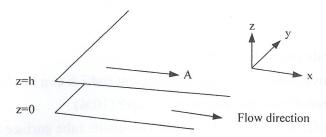


Fig. 1

Equation of continuity:

$$\frac{\partial \rho}{\partial t} + \frac{\partial (\rho v_x)}{\partial x} + \frac{\partial (\rho v_y)}{\partial y} + \frac{\partial (\rho v_z)}{\partial z} = 0.$$

Equation of motion for a constant density and Newtonian fluid:

$$\rho\left(\frac{\partial v_x}{\partial t} + v_x \frac{\partial v_x}{\partial x} + v_y \frac{\partial v_x}{\partial y} + v_z \frac{\partial v_x}{\partial z}\right) = -\frac{\partial p}{\partial x} + \mu\left(\frac{\partial^2 v_x}{\partial x^2} + \frac{\partial^2 v_x}{\partial y^2} + \frac{\partial^2 v_x}{\partial z^2}\right) + \rho g_x,$$

$$\rho\left(\frac{\partial v_y}{\partial t} + v_x \frac{\partial v_y}{\partial x} + v_y \frac{\partial v_y}{\partial y} + v_z \frac{\partial v_y}{\partial z}\right) = -\frac{\partial p}{\partial y} + \mu\left(\frac{\partial^2 v_y}{\partial x^2} + \frac{\partial^2 v_y}{\partial y^2} + \frac{\partial^2 v_y}{\partial z^2}\right) + \rho g_y,$$

$$\rho\left(\frac{\partial v_z}{\partial t} + v_x \frac{\partial v_z}{\partial x} + v_y \frac{\partial v_z}{\partial y} + v_z \frac{\partial v_z}{\partial z}\right) = -\frac{\partial p}{\partial z} + \mu\left(\frac{\partial^2 v_z}{\partial x^2} + \frac{\partial^2 v_z}{\partial y^2} + \frac{\partial^2 v_z}{\partial z^2}\right) + \rho g_z.$$

2. It is known that for fully developed turbulent flow in conduits with Re > 10000, 0.7 < Pr < 160, and L/d > 60, the following Chilton-Colburn analogy can be applied.

$$J_{M} = \frac{f}{2} = J_{H} = \frac{Nu}{Re \ Pr^{1/3}} = J_{D} = \frac{Sh}{Re \ Sc^{1/3}}$$
 (  $f = \frac{\tau_{W}}{\frac{1}{2}\rho u^{2}}$   $Nu = \frac{hd}{k_{c}}$   $Re = \frac{\rho ud}{\mu}$   $Pr = \frac{c_{p}\mu}{k_{c}}$   $Sh = \frac{kd_{h}}{D}$   $Sc = \frac{\mu}{\rho D}$  )

For the following correlation (Eq. (1)) commonly adopted for convective heat transfer through conduit wall in the turbulent flow regime, please suggest a correlation for estimating convective mass transfer coefficient. (10%)

$$Nu = 0.023 Re^{0.8} Pr^{\frac{1}{3}}$$
 (1)

## 4-2

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- 3. A tubular feedwater heater is being designed to heat 0.368 kg/s of water from 21°C to 90°C. Saturated steam at 923 kPa (176°C) is condensing on the outside of the tubes. The tubes are a copper alloy. Each tube has a 2.54 cm outside diameter and a 2.29 cm inside diameter. The water velocity inside the tubes is 0.9 m/s. For the steam outside the tube, the convective heat transfer coefficient can be assumed to be  $13600 \frac{W}{m^2 \text{ K}}$ . The heat transfer resistance of the tube is negligible. (For water,  $C_p = 4.1864 \frac{kJ}{kg \text{ K}}$   $\rho = 1000 \frac{kg}{m^3}$   $\mu = 0.5 \times 10^{-3} \frac{kg}{m \text{ s}}$   $k_c = 0.65 \frac{W}{m \text{ K}}$  Pr=3.37)
- (1) What is the heat transfer rate required? (10%)
- (2) What are the tube side convective heat transfer coefficient (which can be estimated using Eq. (1)) and the overall heat transfer coefficient of the heater? (10%)
- (3) What is the total heat transfer area required in terms of outside tube surface area? (10%)
- 4. For the separation of a mixture using packed bed distillation column, the mass flux of species 'A' across the gas-liquid interface can be written as Eq. (2).
- (1) Please explain Eq. (2) based on two-film theory. A diagram must be sketched with variables marked on it. (10%)
- (2) Please provide the definition and dimension for each variable in Eq. (2). (10%)
- (3) Please derive Eq. (3) and define the variable 'm'. (10%)

$$N_{A} = k_{G}(y_{AG} - y_{Ai})$$

$$= k_{L}(x_{Ai} - x_{AL})$$

$$= K_{OG}(y_{AG} - y_{A}^{*})$$

$$= K_{OL}(x_{A}^{*} - x_{AL})$$
(2)

$$\frac{1}{K_{OG}} = \frac{1}{k_G} + \frac{m}{k_L} \tag{3}$$